

**SA-VPP Power Catamaran 1.1\_User Guide**

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**Version 1.1** : proposes a better estimation of the drag in the low Froude range 0 to 0,4 thanks to the use of the Holtrop-Mennen formulations for the residuary drag in that range (and also valid for  $Lw/Bw > 7$ ) and for the drag of the immersed rear transoms if any.

**SA-VPP Power Catamaran 1.1** (Velocity Prediction Program) makes possible the prediction of the catamaran drag and power to install for speeds in the Froude range 0 to 1. Conditions of validity are :

- Length Beam ratio  $Lw/Bw > 7$
- Length Displacement ratio  $Lw/Dc^{(1/3)} > 6,5$   
(Length, Beam, Displacement here considered for one hull)
- Space Length  $S/Lw$  ratio 0,3 to 0,5

It is a free and open source spreadsheet application, developed on a support itself free and widespread (Open Office Calc 4.0.1) : to open and use an ods file, you have to download Open Office or Libre office according to : <http://www.openthefile.net/extension/ods>

**User guide**

The spreadsheet application includes 2 sheets :

- SA-VPP
- Data storage , where can be stored the various set of input data

**SA-VPP** : includes an User space (input & outputs) followed by an Administrator space (from line 74) where all the computations are carried out. The User space includes :

**The data to input**

Example 3	Lwl (m)	Bwl (m)	Tc (m)	i (°)	Sw (m2)	Disp. (m3)	Cp	At (m2)	LCB (%Lw)
Each Hull >>	15,00	1,60	0,77	9,44	25,40	7,70072	0,565	0,061	45,95
	Hulls axis space S (m)		Saero (m2)	Cx aero	Propulsion efficiency		Cms	Cwp	
Cata >>	6,10		23	0,40	0,54		0,73	0,73	

**For each hull :**

**Lwl (m)** : lenght of waterline

**Bwl (m)** : beam of waterline

**Tc (m)** : hull body draft

**i (°)** : half entry angle of the waterline at bow (neglecting the local rounded shape at the stem)

**Sw (m<sup>2</sup>)** : wetted area of one hull

**Disp. (m<sup>3</sup>)** : displacement of one hull (i.e. half displacement of the catamaran)

**Cp** : hull prismatic coefficient

**LCB (%)** : Longitudinal center of buoyancy, in % of Lwl from aft perpendicular

**At (m<sup>2</sup>)** : immersed part of the transom area at zero speed

**Cms** : midship section coefficient (section / Lw Bw)

**Cwp** : waterplane area coefficient (Sf / Lw Bw)

#### For the catamaran as a whole :

**S (m)** : transversal space between the 2 hulls axis

**Saero (m<sup>2</sup>)** : the frontal section

**Cx aero** : The Cx to take into account for the wind drag computation (to estimate due to the overall shape of the superstructure)

**Propulsion efficiency** : to estimate with the one for the propeller and the one of the mechanical transmission. Example : 0,54 = 0,6 (propeller) x 0,9 (mechanical transmission)

On the right side of the input data is a check of the ratios / necessary conditions :

Conditions :		
> Lw/Bw		
9,38	> 7	
>Lw/Dc <sup>(1/3)</sup>		
7,60	> 6,5	
> S/Lw		
0,41	0,3 to 0,5	

## Output data

, include the main figures and 3 curves :

Ship				Hulls frictional drag Df		Hull residuary drag Dr							Transom drag		Aero drag	D total	> Power net	EHP	Power to install		
	Vb (Knots)	Vb (m/s)	½ Rho Vb²/2	Re hull	Cf hull	Df (kN)	Fn	Holtrop R <sub>w-A</sub> %	Formule 1 Fn < 0,4	Formule 2 0,4 < Fn < 0,6	Formule 3 Fn > 0,6	> Formule all Fn	Dr/mg (%)	K interhull	Dr (kN)	Dtr (kN)	Daero (kN)			Dtot (kN)	EHP (kW)
0,0	0,000		0,0			0,000	0,000	0,000	0,000	-12,329	0,699	0,000	0,000	0,31	0,000	0,000	0,000	0,000	0,00	0,00	0,0
0,8	0,404		83,8	4.25E+006	0,0035	0,015	0,033	0,000	0,001	-10,636	0,875	0,001	0,000	0,31	0,000	0,002	0,001	0,018	0,01	0,01	0,0
1,6	0,809		335,2	8.49E+006	0,0031	0,053	0,067	0,000	0,006	-9,037	1,051	0,006	0,000	0,31	0,000	0,006	0,004	0,062	0,05	0,07	0,1
2,4	1,213		754,1	1.27E+007	0,0029	0,110	0,100	0,000	0,022	-7,531	1,228	0,022	0,000	0,31	0,000	0,012	0,008	0,130	0,16	0,21	0,4
3,1	1,617		1340,7	1.70E+007	0,0027	0,187	0,133	0,001	0,056	-6,118	1,404	0,056	0,001	0,31	0,001	0,017	0,015	0,199	0,25	0,35	0,8
3,9	2,022		2094,8	2.12E+007	0,0026	0,281	0,167	0,004	0,113	-4,798	1,581	0,113	0,004	0,31	0,009	0,020	0,023	0,333	0,67	0,90	1,7
4,7	2,426		3016,6	2.55E+007	0,0026	0,393	0,200	0,018	0,202	-3,572	1,757	0,202	0,018	0,31	0,037	0,019	0,033	0,483	1,17	1,57	2,9
5,5	2,830		4105,9	2.97E+007	0,0025	0,522	0,233	0,052	0,330	-2,438	1,934	0,330	0,052	0,31	0,106	0,014	0,045	0,687	1,95	2,61	4,8
6,3	3,235		5362,8	3.40E+007	0,0025	0,668	0,267	0,111	0,505	-1,398	2,110	0,505	0,155	0,31	0,315	0,002	0,059	1,044	3,38	4,53	8,4
7,1	3,639		6787,3	3.82E+007	0,0024	0,830	0,300	0,188	0,735	-0,452	2,286	0,735	0,370	0,32	0,755	0,000	0,075	1,660	6,04	8,10	15,0
7,9	4,044		8379,4	4.25E+007	0,0024	1,008	0,333	0,337	1,029	0,402	2,463	1,029	0,721	0,32	1,474	0,000	0,092	2,575	10,41	13,96	25,9
8,6	4,448		10139,0	4.67E+007	0,0023	1,202	0,367	0,622	1,394	1,162	2,639	1,394	1,222	0,32	2,502	0,000	0,111	3,816	16,97	22,76	42,1
9,4	4,852		12066,3	5.09E+007	0,0023	1,411	0,400	1,025	1,839	1,829	2,816	1,829	1,829	0,32	3,750	0,000	0,133	5,294	25,69	34,44	63,8
10,4	5,372		14790,5	5.64E+007	0,0023	1,704	0,443		2,543	2,550	3,042	2,550	2,550	0,32	5,221	0,000	0,163	7,087	38,07	51,05	94,5
11,5	5,892		17791,6	6.19E+007	0,0022	2,021	0,486		3,412	3,117	3,269	3,117	3,117	0,31	6,304	0,000	0,196	8,521	50,21	67,33	124,7
12,5	6,412		21069,9	6.73E+007	0,0022	2,363	0,529		4,467	3,529	3,496	3,529	3,529	0,28	6,997	0,000	0,232	9,592	61,50	82,48	152,7
13,5	6,932		24625,1	7.28E+007	0,0022	2,730	0,571		5,726	3,787	3,723	3,787	3,787	0,25	7,351	0,000	0,271	10,352	71,76	96,23	178,2
14,5	7,452		28457,4	7.82E+007	0,0022	3,122	0,614		7,210	3,892	3,950	3,950	3,950	0,23	7,536	0,000	0,313	10,970	81,75	109,62	203,0
15,5	7,971		32566,7	8.37E+007	0,0021	3,537	0,657		8,937	3,842	4,177	4,177	4,177	0,22	7,881	0,000	0,358	11,776	93,87	125,89	233,1
16,5	8,491		36953,0	8.92E+007	0,0021	3,977	0,700		10,930	3,638	4,403	4,403	4,403	0,21	8,261	0,000	0,406	12,644	107,36	143,97	266,6
17,5	9,011		41616,4	9.46E+007	0,0021	4,440	0,743		13,207	3,280	4,630	4,630	4,630	0,21	8,664	0,000	0,458	13,561	122,21	163,88	303,5
18,5	9,531		46556,8	1.00E+008	0,0021	4,927	0,786		15,791	2,768	4,857	4,857	4,857	0,21	9,080	0,000	0,512	14,518	138,38	185,56	343,6
19,5	10,051		51774,3	1.06E+008	0,0021	5,437	0,829		18,701	2,102	5,084	5,084	5,084	0,21	9,501	0,000	0,569	15,507	155,86	209,01	387,1
20,5	10,571		57268,8	1.11E+008	0,0021	5,970	0,871		21,960	1,281	5,311	5,311	5,311	0,21	9,924	0,000	0,630	16,524	174,68	234,24	433,8
21,6	11,091		63040,3	1.16E+008	0,0020	6,527	0,914		25,588	0,307	5,537	5,537	5,537	0,21	10,348	0,000	0,693	17,568	194,84	261,29	483,9
22,6	11,611		69088,8	1.22E+008	0,0020	7,107	0,957		29,608	-0,822	5,764	5,764	5,764	0,21	10,772	0,000	0,760	18,638	216,40	290,19	537,4
23,6	12,131		75414,4	1.27E+008	0,0020	7,709	1,000		34,041	-2,104	5,991	5,991	5,991	0,21	11,196	0,000	0,829	19,734	239,38	321,01	594,5

Df is the frictional drag

Dr is the residuary drag

, where K is the interhulls factor due to waves interference :

$$Dr \text{ cata} = (1+K) (Dr \text{ of the 2 hulls})$$

Dtr is the rear transoms drag

Daero is the aerodynamical drag

Dtot is the total drag

EHP : net propulsion power (= drag x speed)

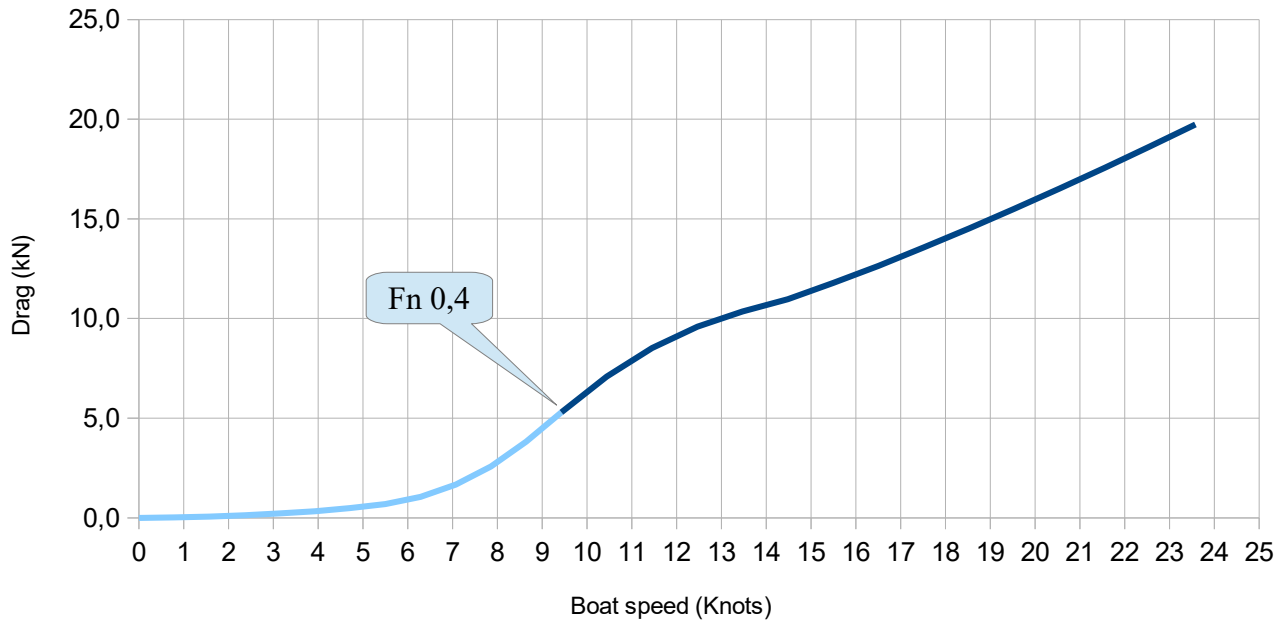
Power to install : taken into account the propulsion efficiency

Example of figures :

..... / .....

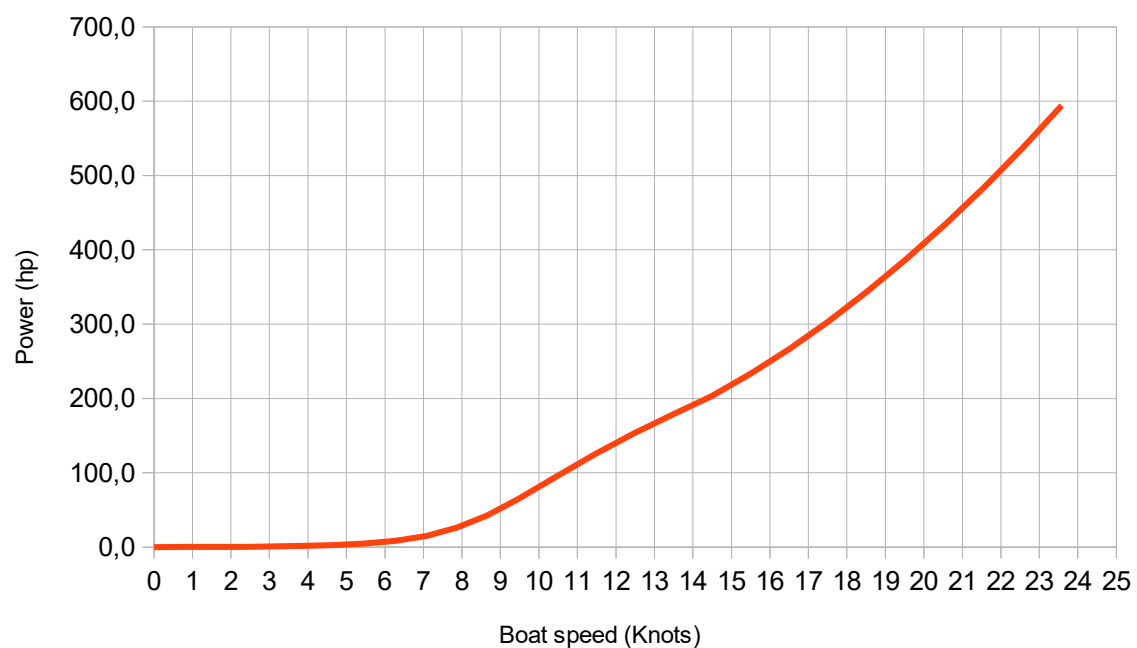
## SA-VPP Power Catamaran - Drag

(calm water, no wind)

>>> here we use two colors to show the smooth connection of the 2 formulations at  $F_n 0,4$ 

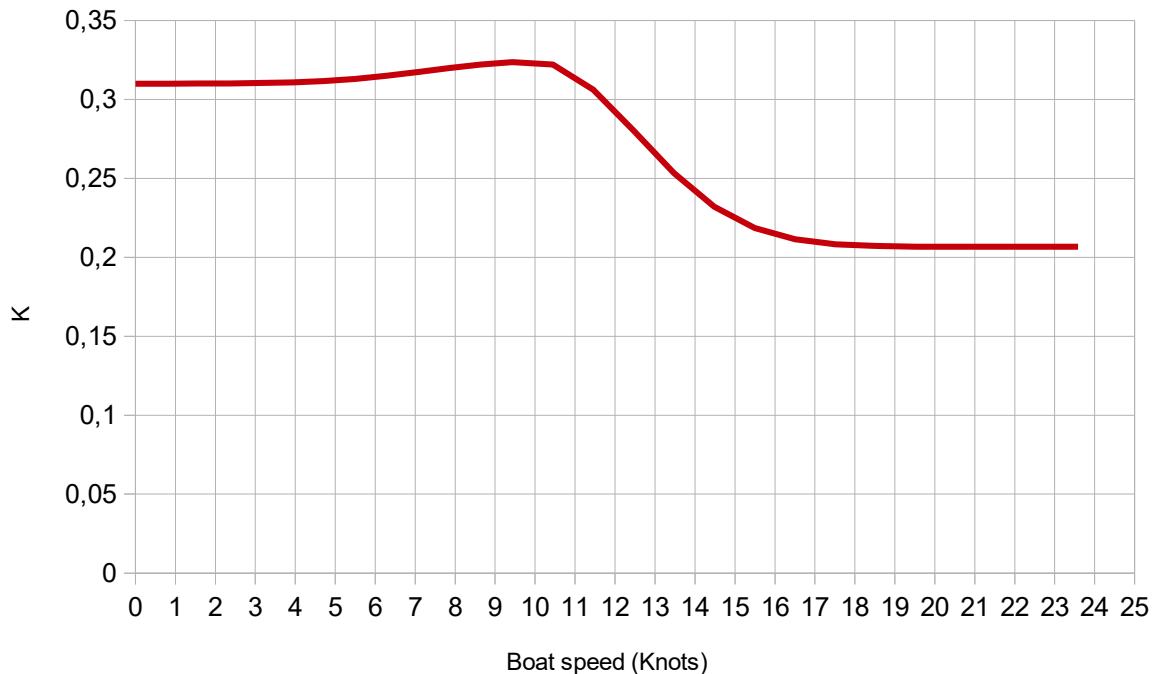
## SA-VPP Power Catamaran - Power to install

(according to the propulsion efficiency input)



### factor K interhulls for the residuary drag Dr

$$>>> D_r \text{ catamaran} = (1+K) (D_r \text{ of the 2 hulls})$$



## Annex

Information about the sources of the computation :

### For the friction drag and the aero drag :

State of the art formulations as for any boat project, for friction it is the ITTC57 formula for  $C_f$  with the Reynolds based on  $0,7 L_w$  as recommended by Larsson & Eliasson in « Principles of Yacht design ». The key points being of course to have a right value of the wetted surface for the design displacement (for friction drag), of the frontal area and its global  $C_x$  (for the aero drag).

### For the residuary drag :

Within this version 1.1, we use :

- Froude 0 to 0,25 : Holtrop-Mennen formulation named « Rw-A » (valid up to Froude 0,4) and hopefully also valid for slender monohull of  $L_w/B_w > 7$
- Froude 0,25 to 0,4 : smooth transition between the 2 formulations
- Froude 0,4 between to 1,0 : Use of the available output data of model tests as reported by :
  - P. van Oossanen in « Resistance prediction of small high-speed displacement vessels » where I picked up the ones for slender hulls (i.e.  $L_w/B_w > 7$ ) in Series 64, SSPA series and NPL series.
  - Molland & al. in « Resistance experiments on a systematic series of high speed catamaran forms – Variation on length-displacement ratio and breadth-draught ratio », the Southampton series.

**Residuary drag for Froude < 0,4 :**

From Holtop-Mennen formulation « Rw-A » in adimensional form (A means for  $Fn < 0,4$ ) :

$$\llcorner \text{Rw-A} \llcorner / D \rho g = c1 \ c2 \ c5 \ \text{Exp}[m1 \ Fn^d + m4 \ \text{Cos}(\lambda Fn^{-2})]$$

with,

$$c1 = 223105 \ c7^3,78613 \ (Tc/Bw)^{1,07961} \ (90-i)^{-1,37565}$$

$$c7 = 0,229577 \ (Bw/Lw)^{0,33333} \quad \text{when } Bw/Lw < 0,11$$

$$c7 = Bw/Lw \quad \text{when } 0,11 < Bw/Lw < 0,25$$

$$c7 = 0,5 - 0,0625 \ (Lw/Bw) \quad \text{when } Bw/Lw > 0,25$$

$$c2 = 1 \ (\text{no bulbous bow taken into account})$$

$$c5 = 1 - 0,8 \ At / (Bw \ Tc \ Cms)$$

$$m1 = 0,0140407 \ (Lw/Tc) - 1,75254 \ (D^{1/3} / Lw) - 4,79323 \ (Bw/Lw) - c16$$

$$c16 = 8,07981 \ Cp - 13,8673 \ Cp^2 + 6,984388 \ Cp^3 \quad \text{when } Cp < 0,8$$

$$c16 = 1,73014 - 0,7067 \ Cp \quad \text{when } Cp > 0,8$$

$$d = -0,9$$

$$m4 = c15 \ 0,4 \ \text{Exp}(-0,034 \ Fn^{-3,29})$$

$$c15 = -1,69385 \quad \text{when } Lw^3/D < 512$$

$$c15 = -1,69385 + (Lw/D^{1/3} - 8)/2,36 \quad \text{when } 512 < Lw^3/D < 1727$$

$$c15 = 0 \quad \text{when } Lw^3/D > 1727$$

$$\lambda = 1,446 \ Cp - 0,03 \ (Lw/Bw) \quad \text{when } Lw/Bw < 12$$

$$\lambda = 1,446 \ Cp - 0,36 \quad \text{when } Lw/Bw > 12$$

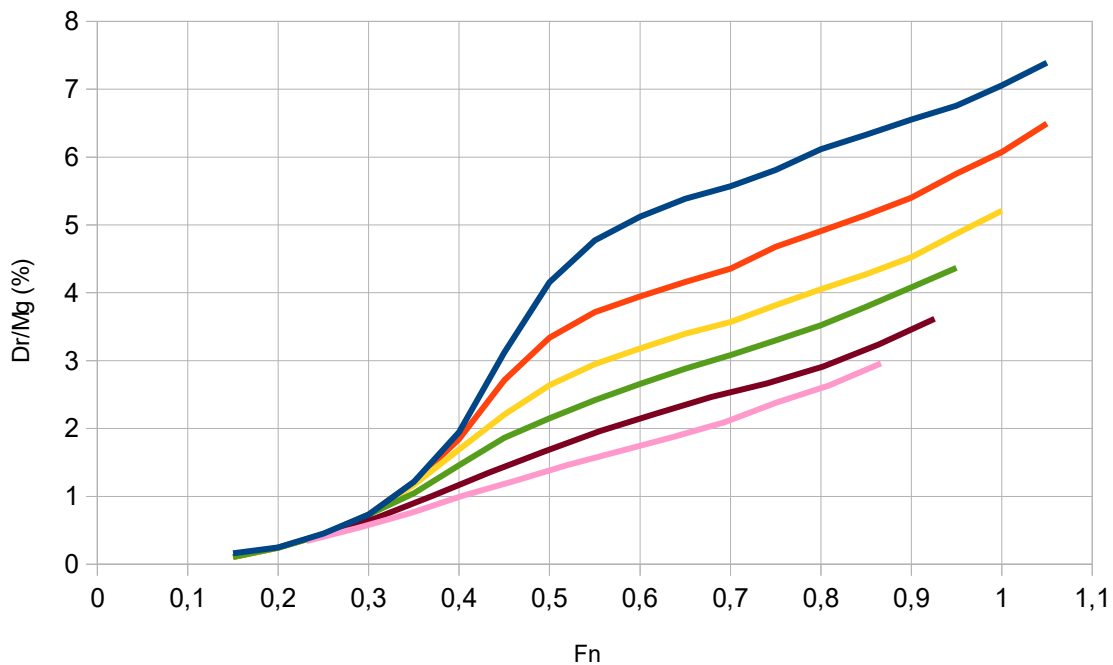
**Residuary drag for Froude 0,4 to 1,0 :**

All the data from the Molland and Oossanen papers were put in same units and presented in the form of adimensional data and curves ready to use.

The key point for narrow monohulls of  $Lw/Bw > 7$ , highlighted by P. Oossanen in his paper and confirmed later on by Molland results was that, quoting P. Oossanen (page 219), «Up to  $Fn$ -value of about 0,9, the results for the 3 B/T values are almost identical, again leading to the observation that in the speed range of  $Fn \ 0,4 - 0,9$  the length-displacement ratio is the only significant parameter », the LD ratio being  $Lw/Dc^{1/3}$  ( $Lw$  waterline length,  $Dc$  displacement of the canoe body hull). The resulting curves are :

### Residuary drag of slender monohull ( $L_w/B_w > 7$ )

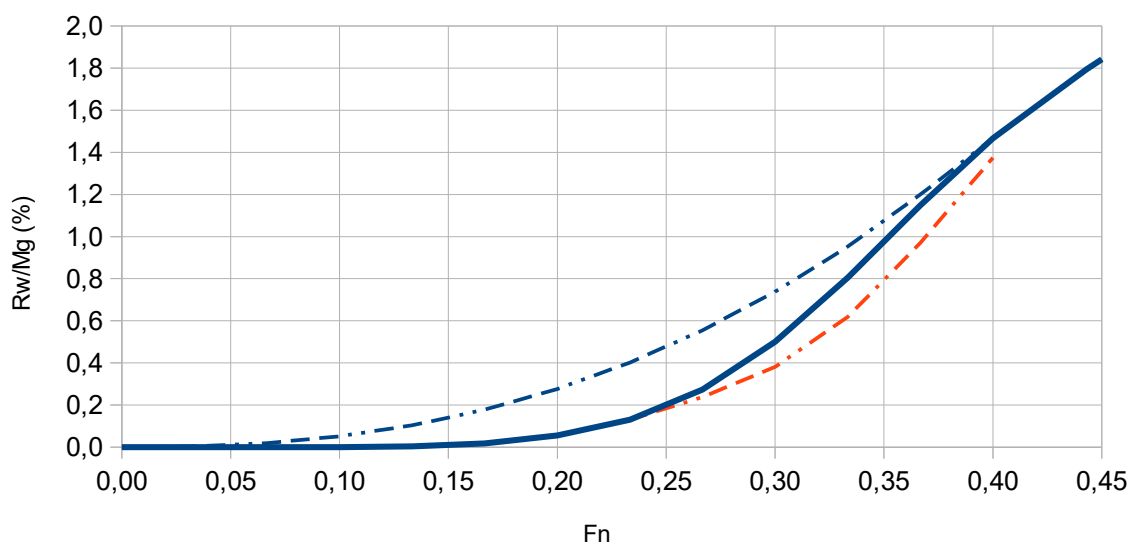
$L_w/D_c^{1/3}$  : Blue : 6,5 ; Red 7,5 ; Yellow 8,5 ; Green 9,5 ; Brown 10,5 ; Pink 11,5 & 12,5  
Averaged/smoothed from Southampton series, Series 64, SSPA series, NPL series



Smooth transition between  $Fn$  0,25 and  $Fn$  0,4, example :

### Residuary drag for slender monohull ( $L/B > 7$ )

Blue continue : SA-VPP Cata  
Blue dashed : former extension for  $Fn < 0,4$  ; Red dashed : Holtrop  $Rw-A$



The other crucial point is the amplification factor  $1+K$  due to interaction between the 2 hulls, fully addressed by Southampton series / Molland experiences in 1994 :

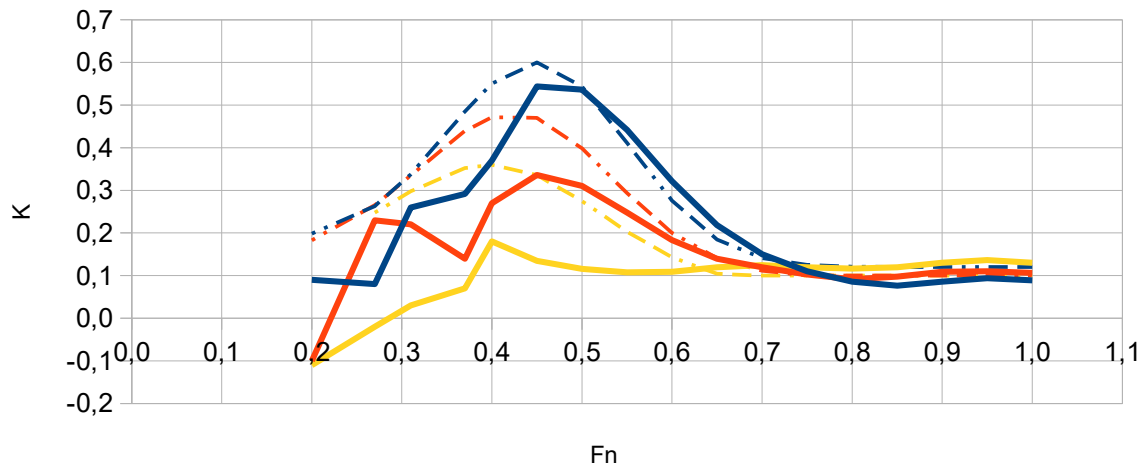
$$\text{Residuary drag : } Dr/Mg (\%) \text{ Cata} = (1+K) * Dr/Mg (\%) \text{ Monohull}$$

Here it is less simple to resume the respective role of  $L_w$ ,  $B_w$ ,  $T_c$  and  $D_c$  combined with  $S$  the space between hull axis >>> the embedded formulation in the VPP try to cover all the cases in a conservative manner. Especially, for Froude 0,2 to 0,4, the experimental results show a lot of steep humps and hollows of that factor whatever the parameters, including for large  $S/L$  (see Fig. 47 to 60 in the Molland paper), which are really challenging to represent by a numerical formulation. So the approach here is mostly to be conservative and to cover as much as possible the humps as the goal is oriented towards the estimation of the necessary power to install. You can see in the following figures the various test results compared with the programmed formulation, so that you can appreciate by yourself the approach :

Amplification factor  $(1+K)$  when  $L_w/D^{(1/3)} = 6,27$  ;  $L_w/B_w = 7$  ;  $B_w/T_c = 2$

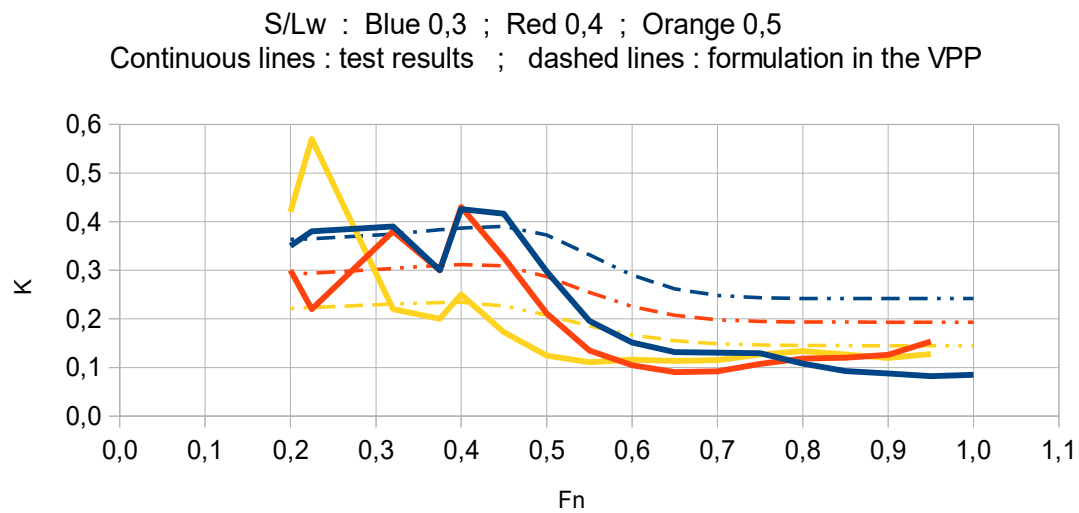
$S/L_w$  : Blue 0,3 ; Red 0,4 ; Orange 0,5

Continuous lines : test results ; dashed lines : formulation in the VPP

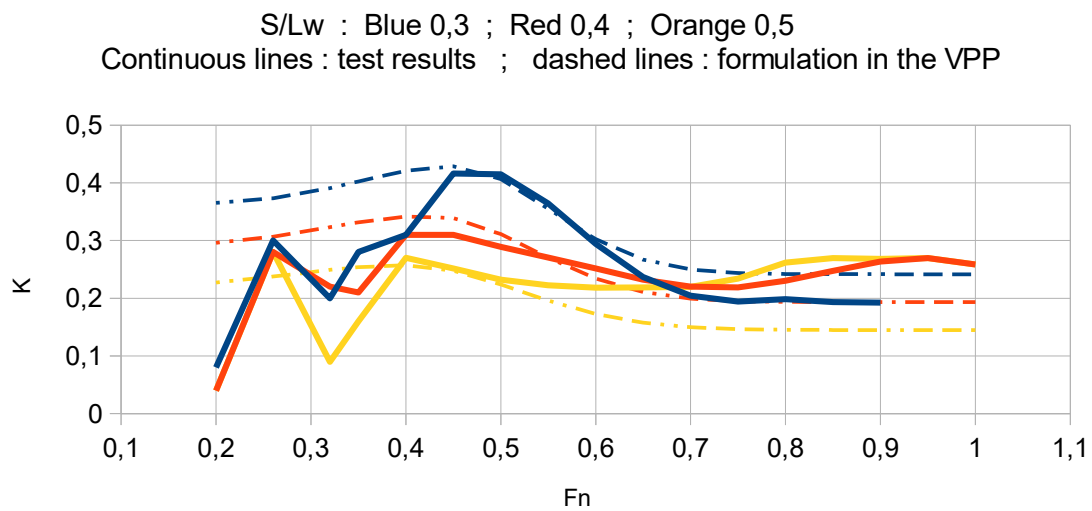




Amplification factor ( $1+K$ ) when  $Lw/D^{(1/3)} = 7,4$  ;  $Lw/Bwl = 10,4$  ;  $Bwl/Tc = 1,5$



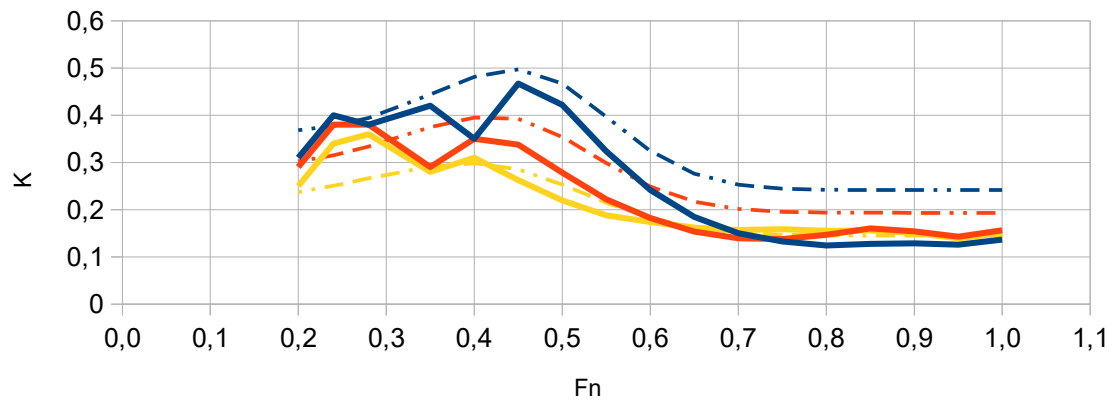
Amplification factor ( $1+K$ ) when  $Lw/D^{(1/3)} = 7,4$  ;  $Lw/Bwl = 9$  ;  $Bwl/Tc = 2$



Amplification factor ( $1+K$ ) when  $Lw/D^{(1/3)} = 7,4$  ;  $Lw/Bwl = 8$  ;  $Bwl/Tc = 2,5$

$S/Lw$  : Blue 0,3 ; Red 0,4 ; Orange 0,5

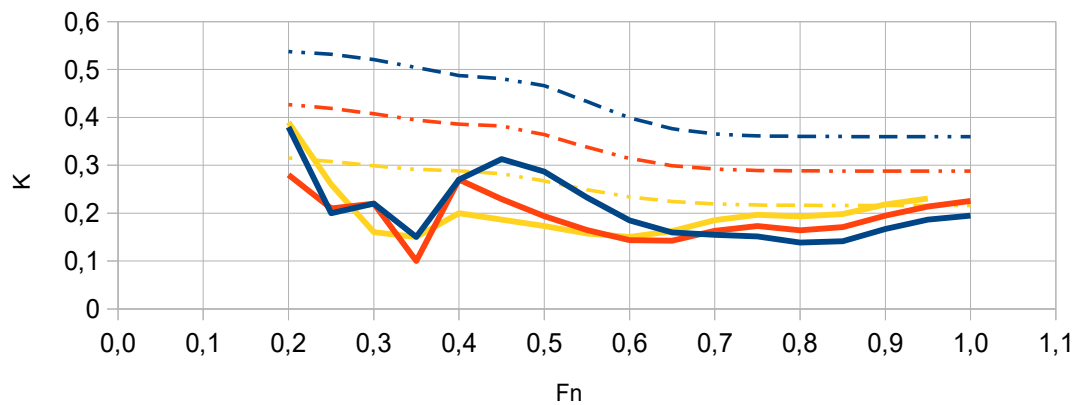
Continuous lines : test results ; dashed lines : formulation in the VPP



Amplification factor ( $1+K$ ) when  $Lw/D^{(1/3)} = 8,5$  ;  $Lw/Bwl = 12,8$  ;  $Bwl/Tc = 1,5$

$S/Lw$  : Blue 0,3 ; Red 0,4 ; Orange 0,5

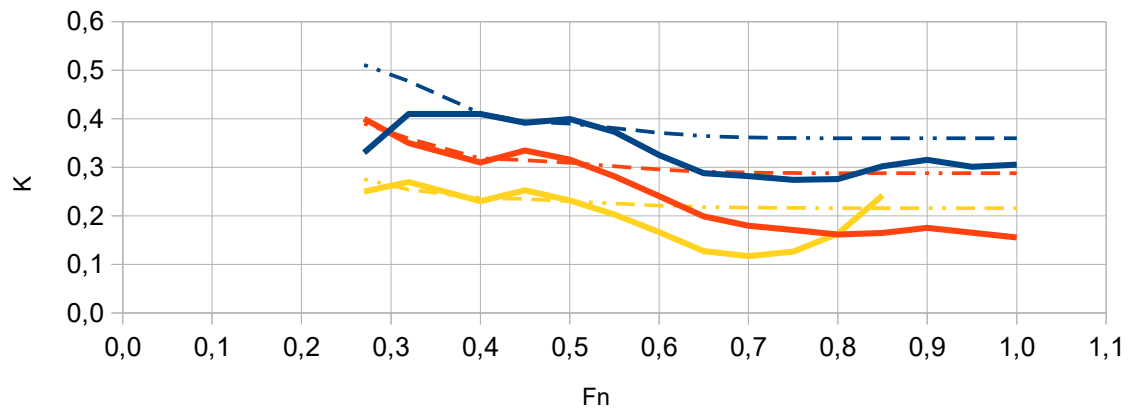
Continuous lines : test results ; dashed lines : formulation in the VPP



Amplification factor  $(1+K)$  when  $Lw/D^{(1/3)} = 8,5$  ;  $Lwl/Bwl = 11$  ;  $Bwl/Tc = 2$

S/Lw : Blue 0,3 ; Red 0,4 ; Orange 0,5

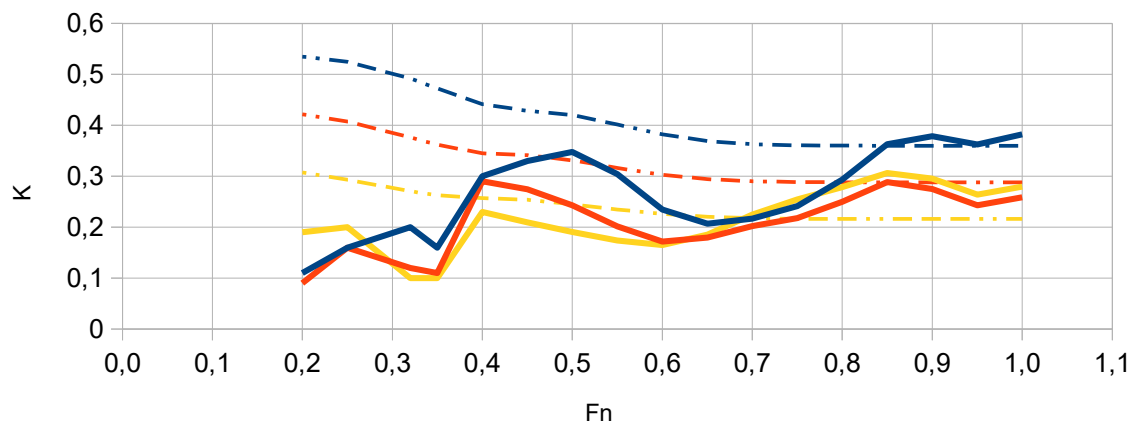
Continuous lines : test results ; dashed lines : formulation in the VPP



Amplification factor  $(1+K)$  when  $Lw/D^{(1/3)} = 8,5$  ;  $Lwl/Bwl = 9$  ;  $Bwl/Tc = 2,5$

S/Lw : Blue 0,3 ; Red 0,4 ; Orange 0,5

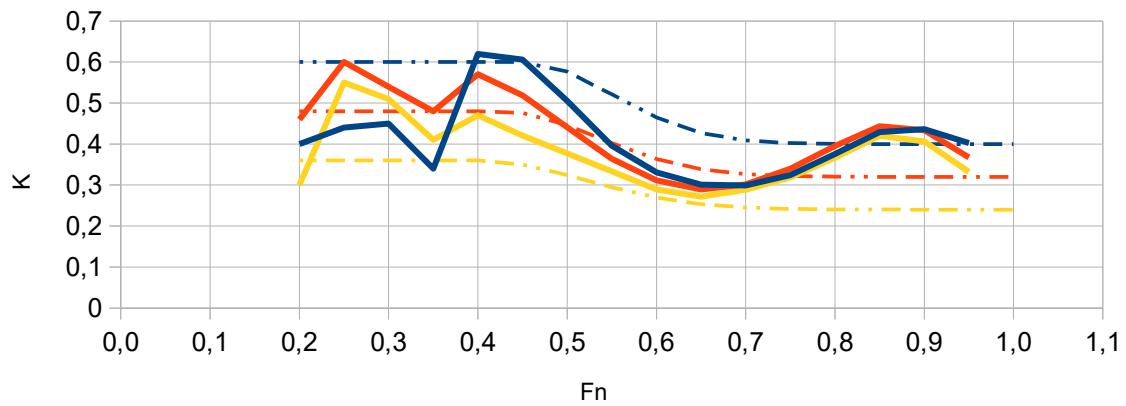
Continuous lines : test results ; dashed lines : formulation in the VPP



Amplification factor ( $1+K$ ) with  $L_w/D^{(1/3)} = 9,5$  ;  $L_w/B_w = 15,1$  ;  $B_w/T_c = 1,5$

$S/L_w$  : Blue 0,3 ; Red 0,4 ; Orange 0,5

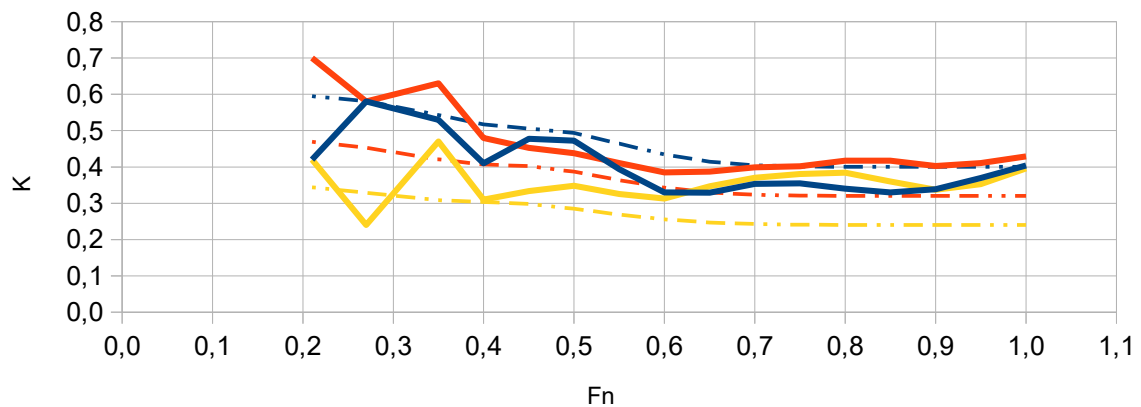
Continuous lines : test results ; dashed lines : formulation in the VPP



Amplification factor ( $1+K$ ) when  $L_w/D^{(1/3)} = 9,5$  ;  $L_w/B_w = 13,1$  ;  $B_w/T_c = 2$

$S/L_w$  : Blue 0,3 ; Red 0,4 ; Orange 0,5

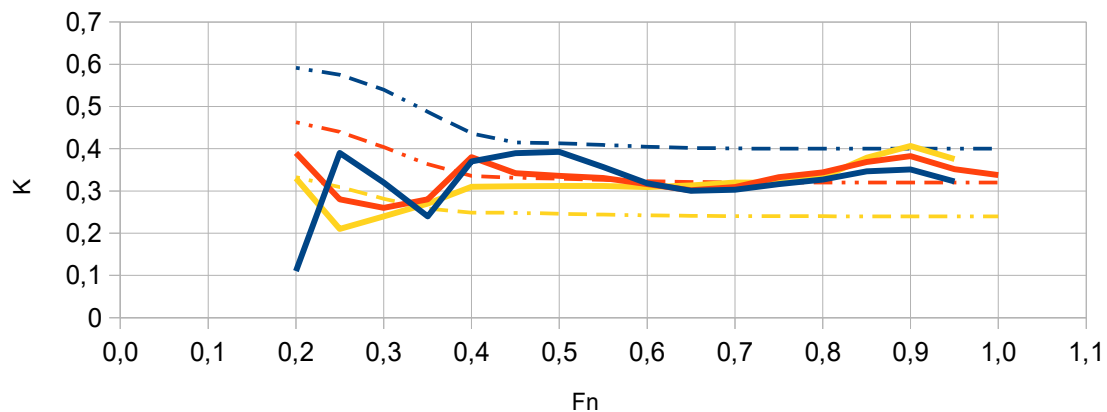
Continuous lines : test results ; dashed lines : formulation in the VPP



Amplification factor  $(1+K)$  when  $Lw/D^{(1/3)} = 9,5$  ;  $Lw/Bwl = 11,7$  ;  $Bwl/Tc = 2,5$

$S/Lw$  : Blue 0,3 ; Red 0,4 ; Orange 0,5

Continuous lines : test results ; dashed lines : formulation in the VPP



Why no longer reliable for  $F_n > 0,9$  -1,0 ? : because then the hulls are no longer primarily in displacement mode, dynamic lift forces occur, and then a lot of parameters like sections shape (circular or in U),  $Bw/Tc$ , trim, ... can change the resulting drag.

**Rear transom drag** (additional pressure resistance due to the immersed transom)

From Holtrop-Mennen formulation :

$$R_{tr} = 0,5 \rho V^2 A_t c_6$$

with ,

$$c_6 = 0,2 (1 - 0,2 F_{nt}) \quad \text{when } F_{nt} < 5$$

$$c_6 = 0 \quad \text{when } F_{nt} > 5$$

$$F_{nt} = V / [2 g A_t / (Bw + Bw C_{wp})]^{0,5} \quad \text{Froude number based on transom immersion}$$