

SA-VPP Power Catamaran 1.0_User Guide

Jean-François Masset – June 2021

contact : jfcmasset@outlook.fr

SA-VPP Power Catamaran (Velocity Prediction Program) makes possible the prediction of the catamaran drag and power to install for speeds in the Froude range 0,3 to 1,0. Conditions of validity are :

- Length Beam ratio $Lw/Bw > 7$
- Length Displacement ratio $Lw/Dc^{(1/3)} > 6,5$
(Length, Beam, Displacement here considered for one hull)
- Space Length S/Lw ratio 0,3 to 0,5

It is a free and open source spreadsheet application, developed on a support itself free and widespread (Open Office Calc 4.0.1) : to open and use an ods file, you have to download Open Office or Libre office according to : <http://www.openthefile.net/extension/ods>

User guide

The spreadsheet application includes 2 sheets :

- SA-VPP
- Data storage , where can be stored the various set of input data

SA-VPP : includes an User space (input & outputs) followed by an Administrator space (from line 74) where all the computations are carried out. The User space includes :

The data to input

Example 3	Lwl (m)	Bwl (m)	Sw (m2)	Disp. (m3)
Each Hull >>	14,75	1,70	28,58	8,42930
	Hulls axis space S (m)	Saero (m2)	Cx aero	Propulsion efficiency
Cata >>	5,20	23	0,40	0,54

First line, for each hull :

Lwl (m) : lenght of waterline

Bwl (m) : beam of waterline

Sw (m2) : wetted area of one hull

Disp. (m3) : displacement of one hull (i.e. half displacement of the catamaran)

Second line, for the catamaran as a whole :

S (m) : transversal space between the 2 hulls axis

Saero (m2) : the frontal section

Cx aero : The Cx to take into account for the wind drag computation (to estimate due to the overall shape of the superstructure)

Propulsion efficiency : to estimate with the one for the propeller and the one of the mechanical transmission. Example : 0,54 = 0,6 (propeller) x 0,9 (mechanical transmission)

On the right side of the input data is a check of the ratios / necessary conditions :

Conditions :

$$\begin{aligned} &>>> Lw/Bw \\ &8,68 > 7 \\ &>>> Lw/Dc^{(1/3)} \\ &7,25 > 6,5 \end{aligned}$$

Output data

, include the main figures and 3 curves :

Ship	Hulls frictional drag Df			Hull residuary drag Dr					Aero drag		D total	> Power net	EHP	Power to instal			
Vb (Knots)	Vb (m/s)	½ Rho Vb²2	Re hull	Cf hull	Df (kN)	Fn	Formule 1 Fn < 0,4	Formule 2 0,4< Fn< 0,6	Formule 3 Fn > 0,6	> Dr/mg (%)	K interhull	Dr (kN)	Daero (kN)	Dtot (kN)	EHP (kW)	EHP (hp)	Power to instal (hp)
7,0	3,609	6674,2	3,73E+007	0,0024	0,922	0,300	0,735	-0,782	2,685	0,735	0,29	1,601	0,073	2,597	9,37	12,57	23,3
8,0	4,135	8762,7	4,27E+007	0,0024	1,185	0,344	1,146	0,498	2,912	1,146	0,33	2,592	0,096	3,873	16,02	21,48	39,8
9,1	4,661	11135,2	4,81E+007	0,0023	1,478	0,388	1,693	1,591	3,139	1,693	0,38	3,958	0,122	5,559	25,91	34,75	64,3
10,1	5,188	13791,6	5,36E+007	0,0023	1,801	0,431	2,400	2,498	3,367	2,498	0,40	5,927	0,152	7,880	40,88	54,82	101,5
11,1	5,714	16731,8	5,90E+007	0,0023	2,154	0,475	3,289	3,217	3,594	3,217	0,39	7,566	0,184	9,904	56,59	75,88	140,5
12,1	6,240	19955,9	6,44E+007	0,0022	2,535	0,519	4,384	3,750	3,821	3,750	0,35	8,565	0,219	11,319	70,63	94,72	175,4
13,2	6,766	23463,9	6,99E+007	0,0022	2,945	0,563	5,709	4,096	4,049	4,096	0,30	9,009	0,258	12,212	82,63	110,81	205,2
14,2	7,293	27255,8	7,53E+007	0,0022	3,383	0,606	7,288	4,255	4,276	4,276	0,25	9,094	0,300	12,776	93,17	124,95	231,4
15,2	7,819	31331,5	8,07E+007	0,0021	3,849	0,650	9,147	4,227	4,503	4,503	0,23	9,360	0,344	13,554	105,97	142,11	263,2
16,2	8,345	35691,2	8,62E+007	0,0021	4,343	0,694	11,312	4,012	4,731	4,731	0,21	9,712	0,392	14,448	120,57	161,68	299,4
17,2	8,871	40334,7	9,16E+007	0,0021	4,865	0,738	13,809	3,611	4,958	4,958	0,20	10,125	0,443	15,433	136,91	183,60	340,0
18,3	9,398	45262,1	9,70E+007	0,0021	5,414	0,781	16,664	3,022	5,186	5,186	0,20	10,569	0,498	16,481	154,88	207,69	384,6
19,3	9,924	50473,4	1,02E+008	0,0021	5,989	0,825	19,905	2,247	5,413	5,413	0,20	11,027	0,555	17,571	174,38	233,84	433,0
20,3	10,450	55968,6	1,08E+008	0,0021	6,592	0,869	23,558	1,285	5,640	5,640	0,20	11,489	0,615	18,696	195,38	262,00	485,2
21,3	10,976	61747,7	1,13E+008	0,0020	7,222	0,913	27,652	0,136	5,868	5,868	0,20	11,951	0,679	19,852	217,91	292,21	541,1
22,4	11,503	67810,6	1,19E+008	0,0020	7,878	0,956	32,214	-1,200	6,095	6,095	0,20	12,414	0,746	21,038	241,99	324,51	600,9
23,4	12,029	74157,5	1,24E+008	0,0020	8,560	1,000	37,274	-2,723	6,322	6,322	0,20	12,877	0,815	22,253	267,68	358,96	664,7

Df is the frictional drag

Dr is the residuary drag

, where K is the amplification factor due to hulls interference :

$$Dr \text{ cata} = (1+K) (Dr \text{ of the 2 hulls})$$

Daero is the aerodynamical drag

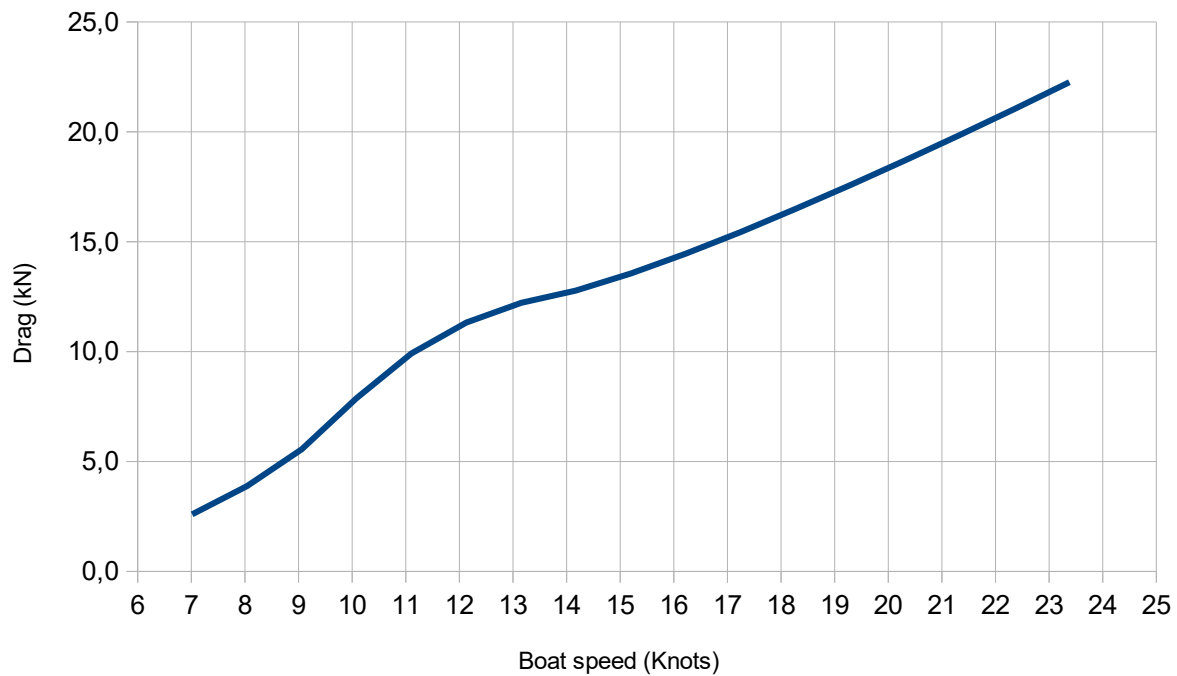
Dtot is the total drag

EHP : net propulsion power (= drag x speed)

Power to install : taken into account the propulsion efficiency

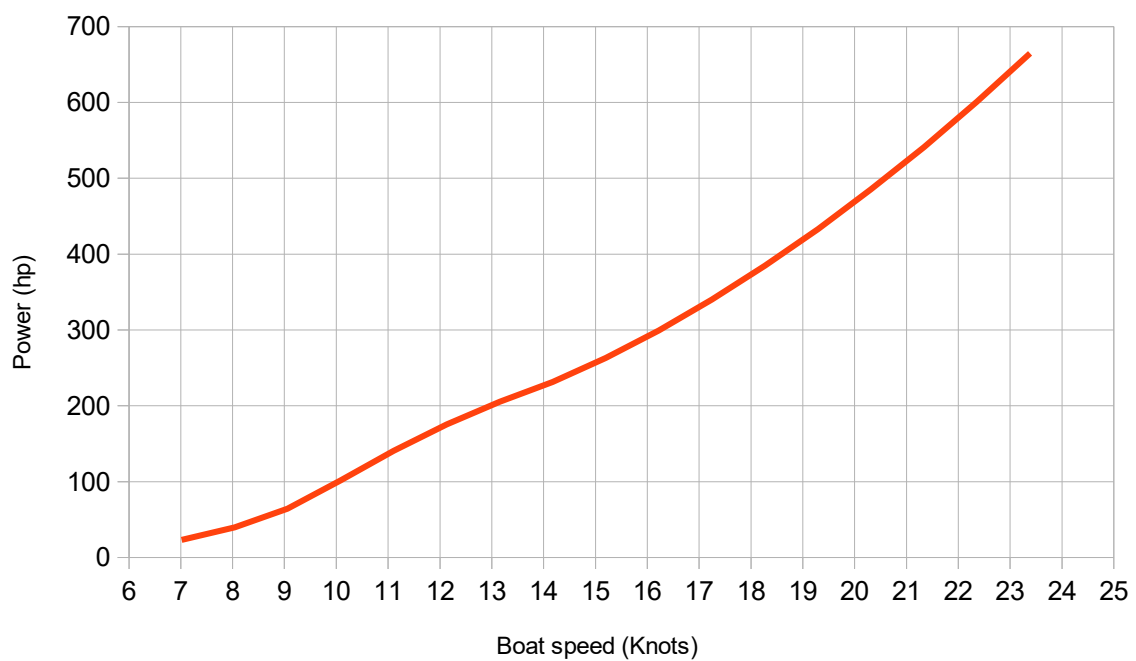
SA-VPP Power Catamaran - Drag

(calm water, no wind)



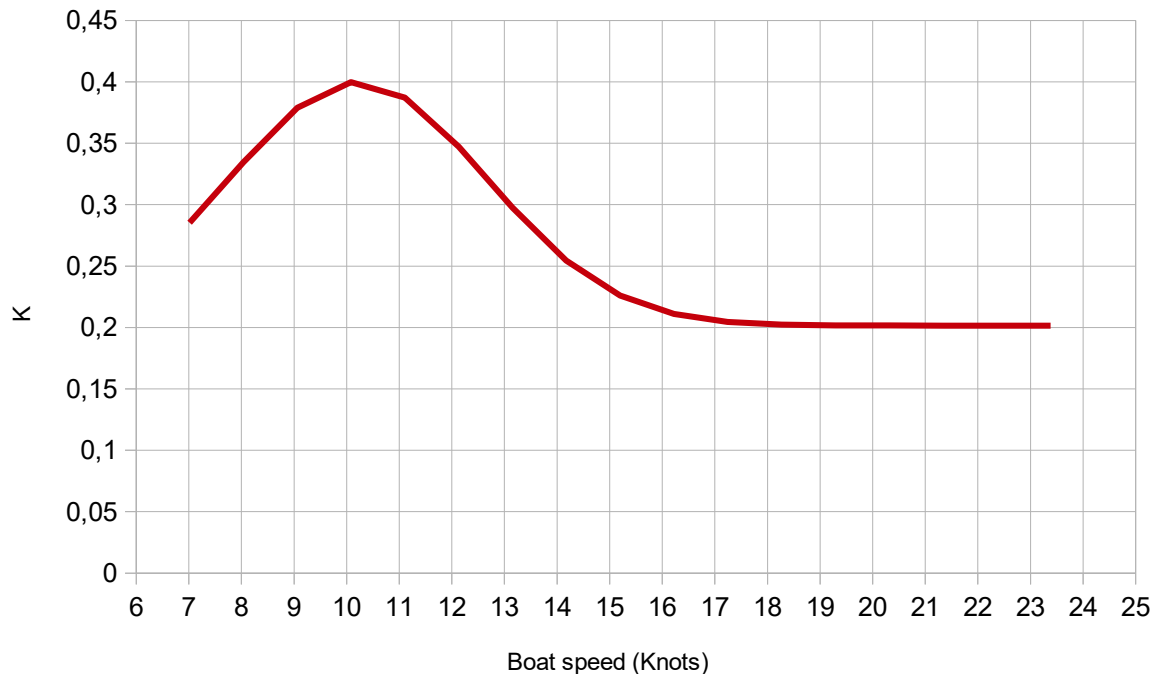
SA-VPP Power Catamaran - Power to install

(according to the propulsion efficiency input)



Amplification factor K interhulls for the residuary drag D_r

$$D_r = (1+K) (D_r \text{ of the 2 hulls})$$



Annex

Information about the sources of the computation :

For the friction drag and the aero drag :

State of the art formulations as for any boat project, for friction it is the ITTC57 formula for C_f with the Reynolds based on $0,7 L_w$ as recommended by Larsson & Eliasson in « Principles of Yacht design ». The key points being of course to have a right value of the wetted surface for the design displacement (for friction drag), of the frontal area and its global C_x (for the aero drag).

For the residuary drag :

Use of the available output data of model tests as reported by :

- P. van Oossanen in « Resistance prediction of small high-speed displacement vessels » where I picked up the ones for slender hulls (i.e. $L_w/B_w > 7$) in Series 64, SSPA series and NPL series.
- Molland & al. in « Resistance experiments on a systematic series of high speed catamaran forms – Variation on length-displacement ratio and breadth-draught ratio », the Southampton series.

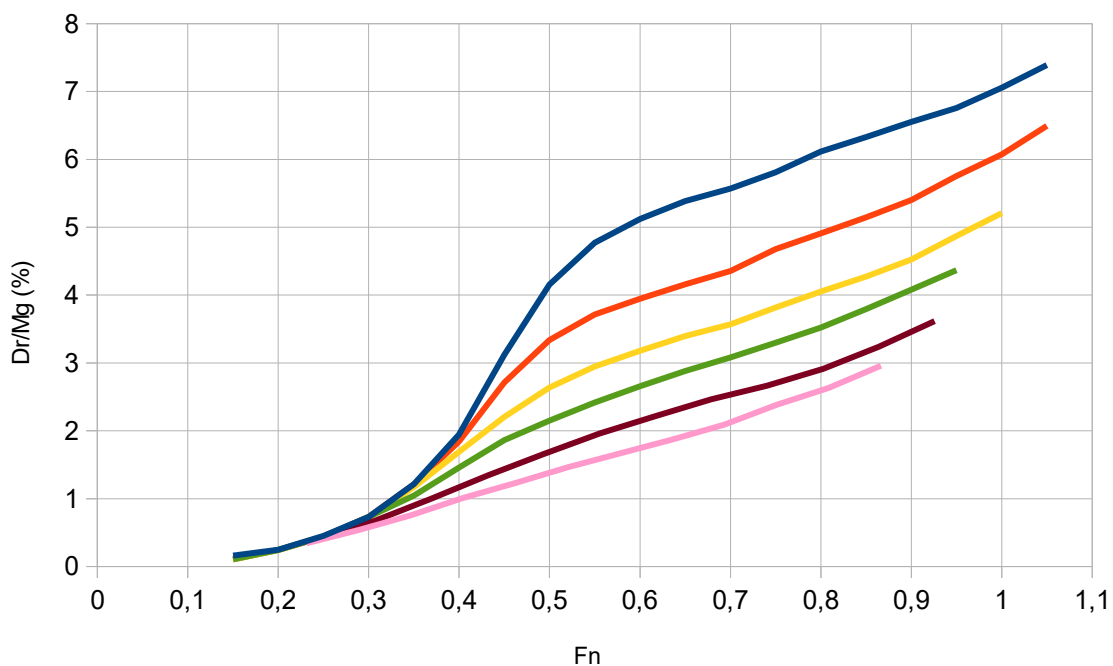
All the data from these papers were put in same units and presented in the form of adimensional data and curves ready to use.

The key point for narrow monohulls of $L_w/B_w > 7$, highlighted by P. Oossanen in his paper and confirmed later on by Molland results was that, quoting P. Oossanen (page 219), «Up to Fn -value of about 0,9, the results for the 3 B/T values are almost identical, again leading to the observation

that in the speed range of F_n 0,4 – 0,9 the length-displacement ratio is the only significant parameter », the LD ratio being $L_w/D_c^{1/3}$ (L_w waterline length, D_c displacement of the canoe body hull). The resulting curves are :

Residuary drag of slender monohull ($L_w/B_w > 7$)

$L_w/D_c^{1/3}$: Blue : 6,5 ; Red 7,5 ; Yellow 8,5 ; Green 9,5 ; Brown 10,5 ; Pink 11,5 & 12,5
Averaged/smoothed from Southampton series, Series 64, SSPA series, NPL series

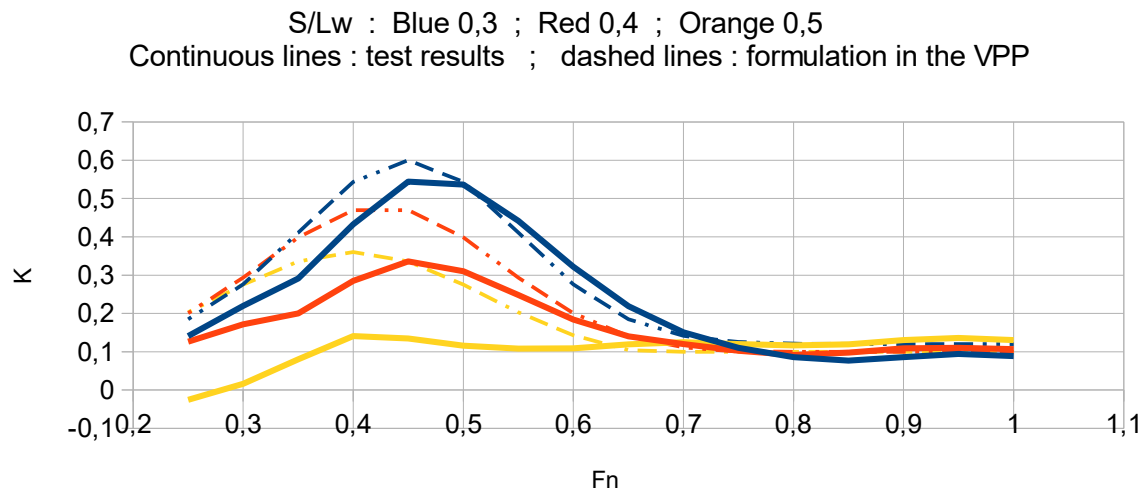


The other crucial point is the amplification factor $1+K$ due to interaction between the 2 hulls, fully addressed by Southampton series / Molland experiences in 1994 :

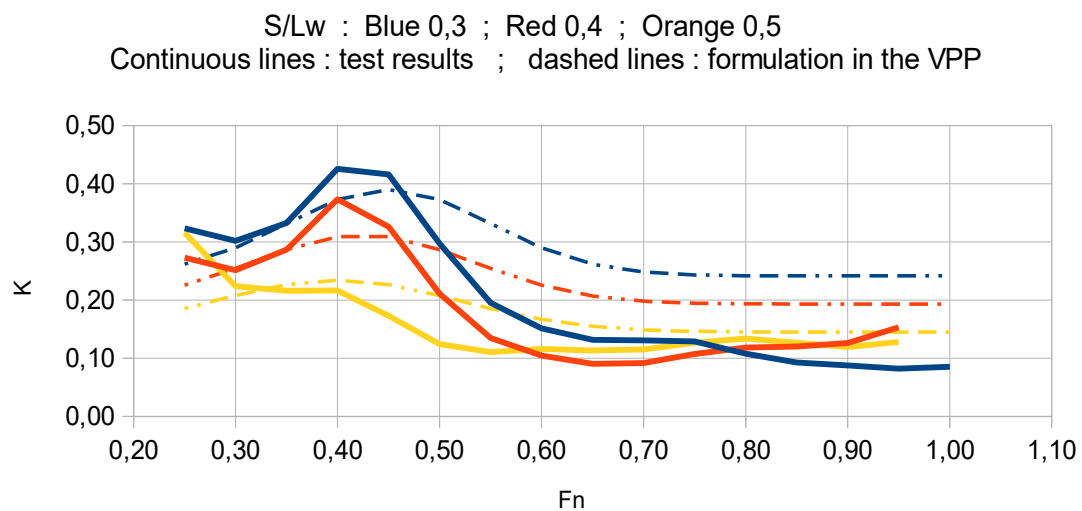
$$\text{Residuary drag : } Dr/Mg (\%) \text{ Cata} = (1+K) * Dr/Mg (\%) \text{ Monohull}$$

Here it is less simple to resume the respective role of L_w , B_w , T_c and D_c combined with S the space between hull axis >>> the embedded formulation in the VPP try to cover all the cases in a conservative manner. You can see in the following figures the various test results compared with the programmed formulation, so that you can appreciate by yourself the approach :

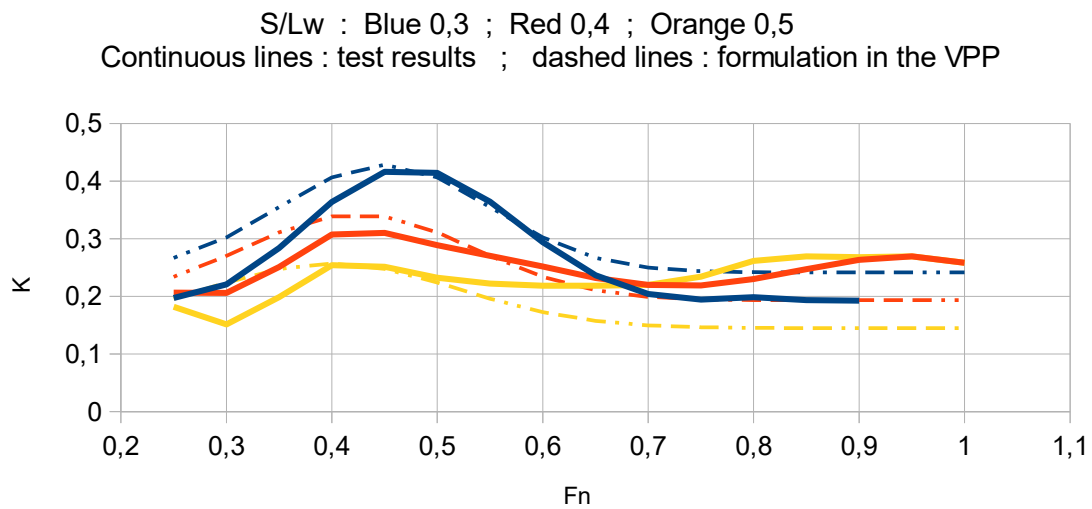
Amplification factor ($1+K$) when $L_w/D^{(1/3)} = 6,27$; $L_w/B_w = 7$; $B_w/T_c = 2$



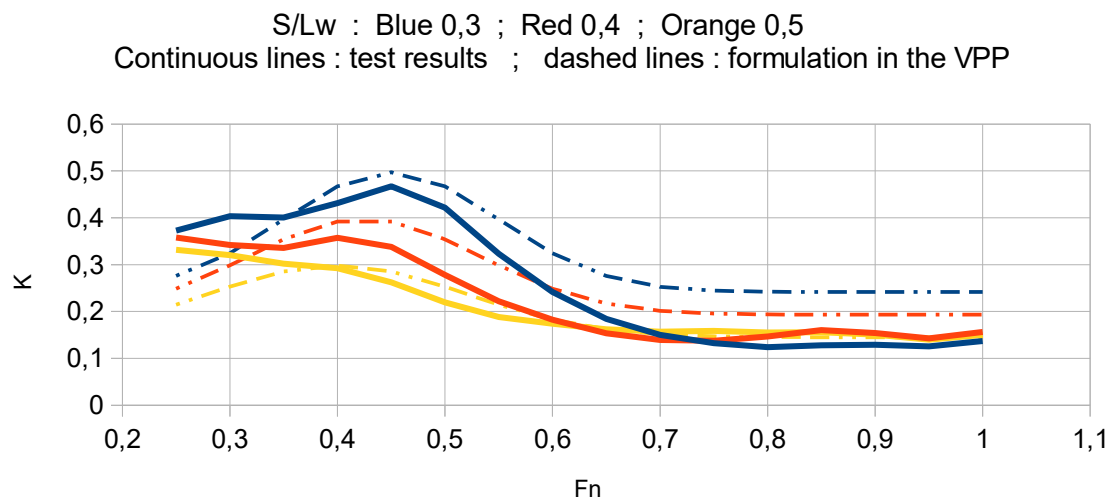
Amplification factor ($1+K$) when $L_w/D^{(1/3)} = 7,4$; $L_w/B_w = 10,4$; $B_w/T_c = 1,5$



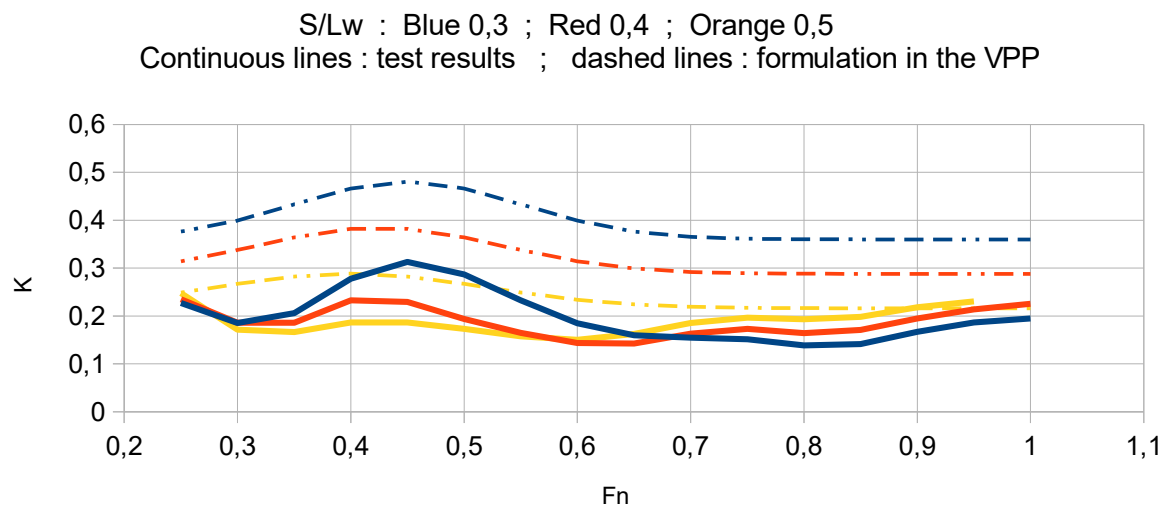
Amplification factor $(1+K)$ when $Lw/D^{(1/3)} = 7,4$; $Lwl/Bwl = 9$; $Bwl/Tc = 2$



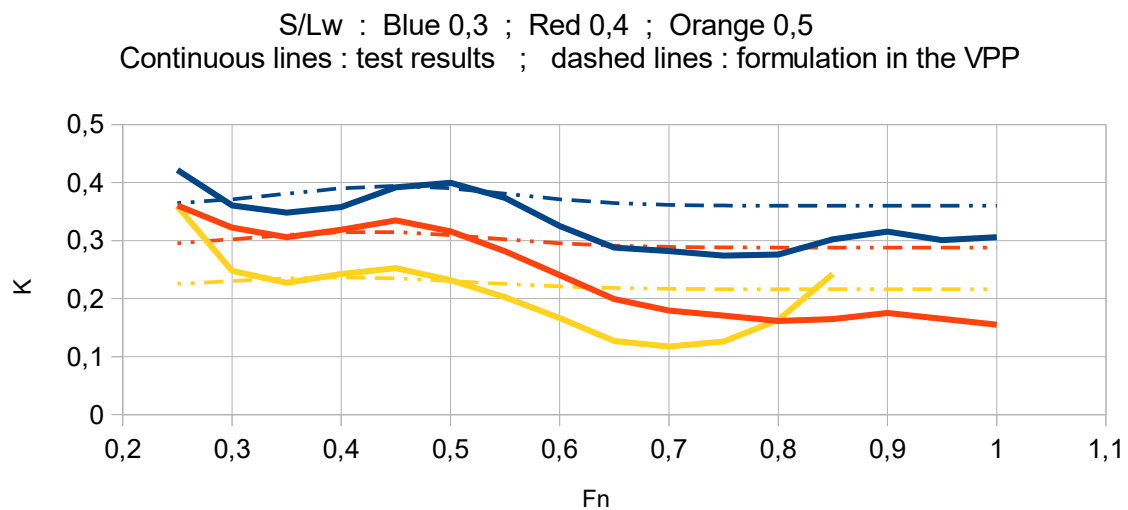
Amplification factor $(1+K)$ when $Lw/D^{(1/3)} = 7,4$; $Lwl/Bwl = 8$; $Bwl/Tc = 2,5$



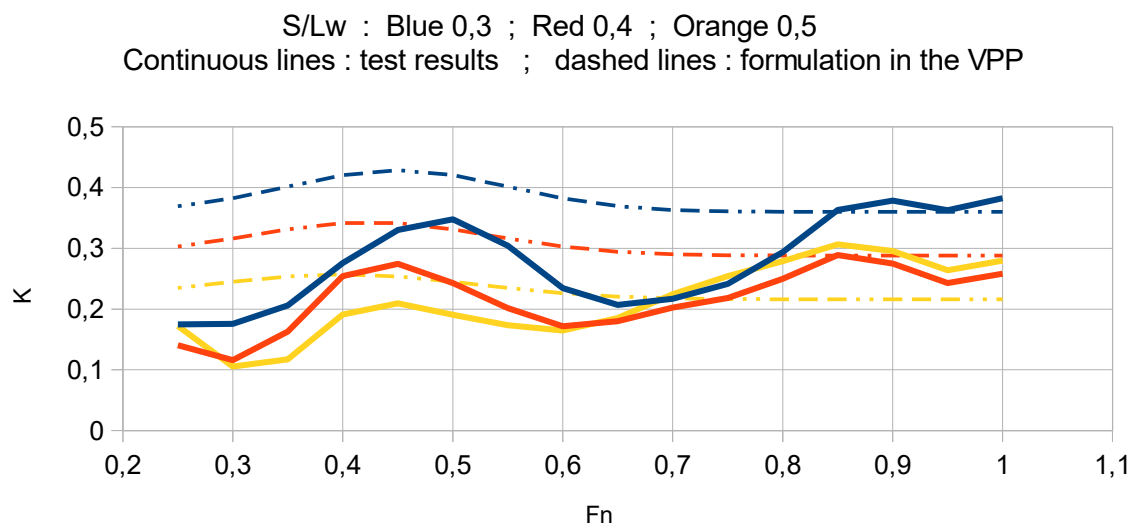
Amplification factor ($1+K$) when $Lw/D^{(1/3)} = 8,5$; $Lw/Bwl = 12,8$; $Bwl/Tc = 1,5$



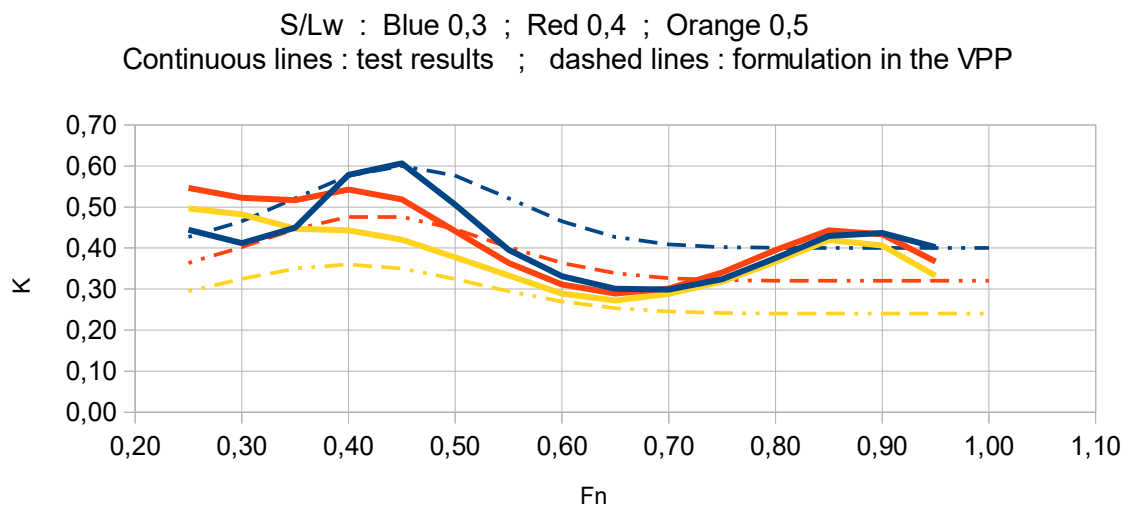
Amplification factor ($1+K$) when $Lw/D^{(1/3)} = 8,5$; $Lw/Bwl = 11$; $Bwl/Tc = 2$



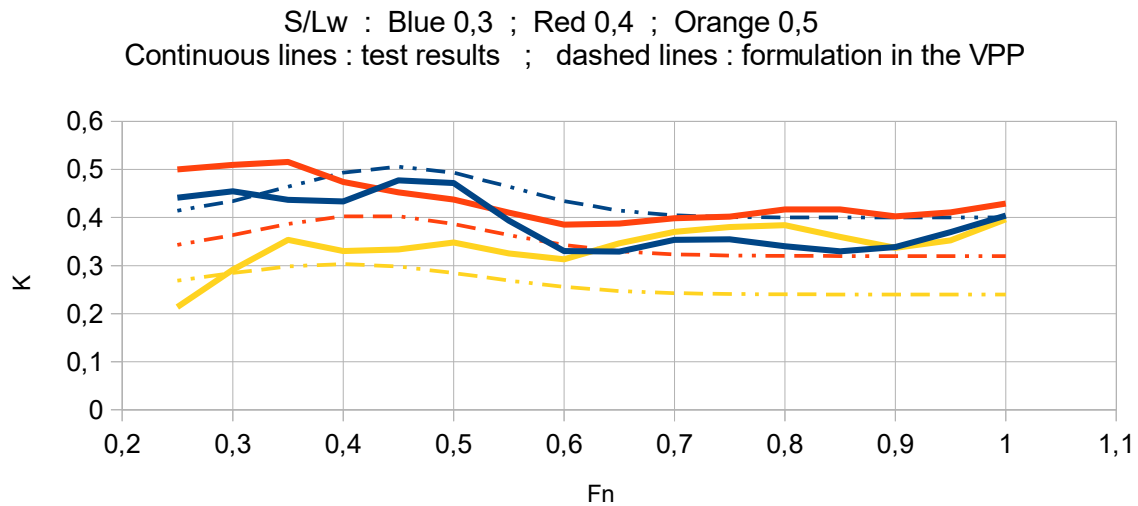
Amplification factor ($1+K$) when $Lw/D^{(1/3)} = 8,5$; $Lwl/Bwl = 9$; $Bwl/Tc = 2,5$



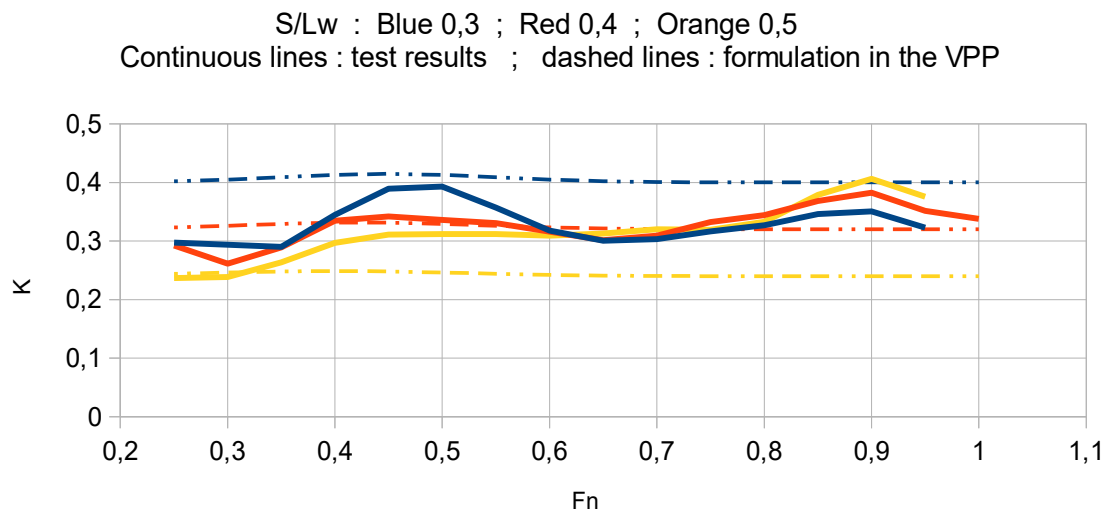
Amplification factor ($1+K$) with $Lw/D^{(1/3)} = 9,5$; $Lwl/Bwl = 15,1$; $Bwl/Tc = 1,5$



Amplification factor ($1+K$) when $Lw/D^{(1/3)} = 9,5$; $Lw/Bwl = 13,1$; $Bwl/Tc = 2$



Amplification factor ($1+K$) when $Lw/D^{(1/3)} = 9,5$; $Lw/Bwl = 11,7$; $Bwl/Tc = 2,5$



Why no longer reliable for $Fn > 0,9 - 1,0$? : because then the hulls are no longer primarily in displacement mode, dynamic lift forces occur, and then a lot of parameters like sections shape (circular or in U), Bw/Tc , trim, ... can change the resulting drag.

Why possibly inaccurate for $Fn < 0,3 - 0,4$? : because the wave fields and their interaction are more complicated, but it should be noted that such speed is usually not a design objective except for slow by purpose catamaran.